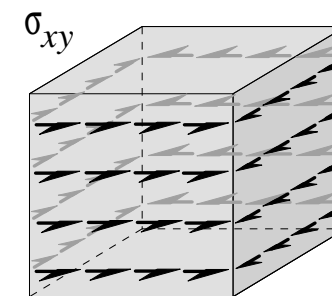
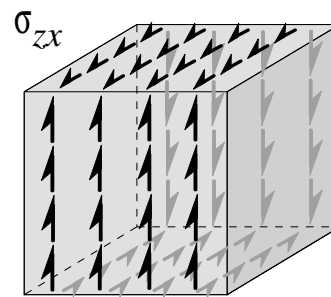
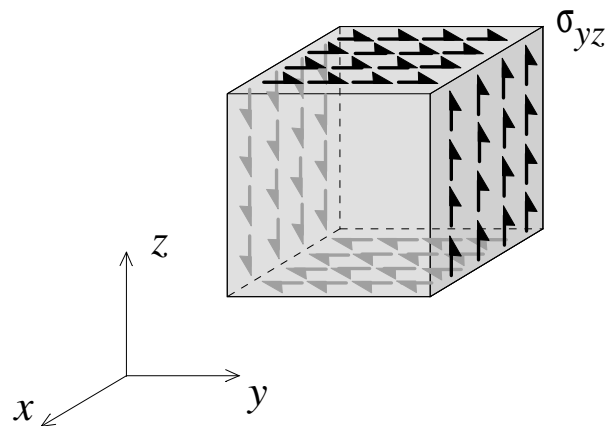
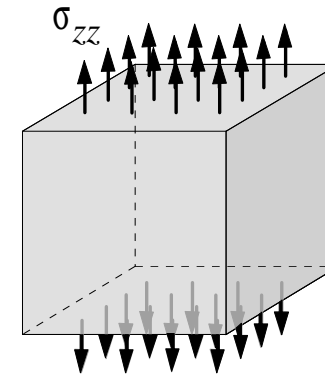
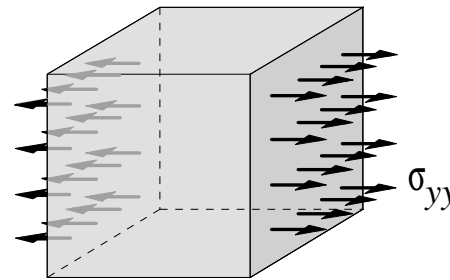
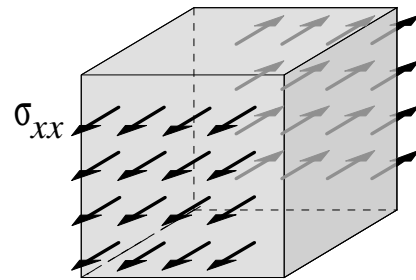
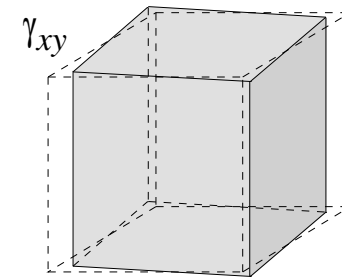
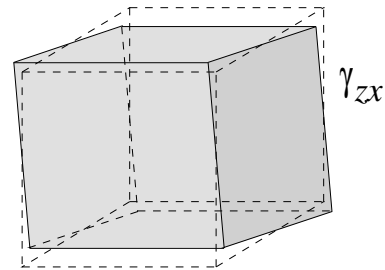
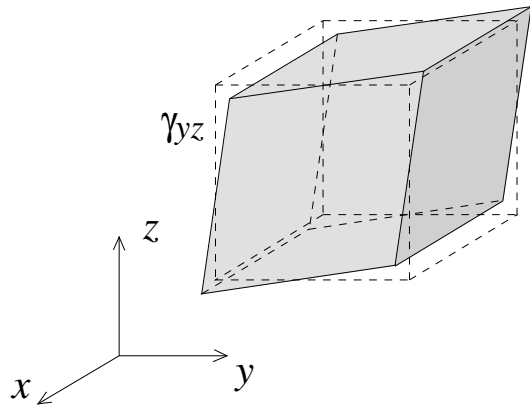
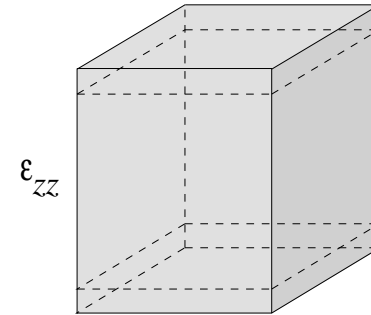
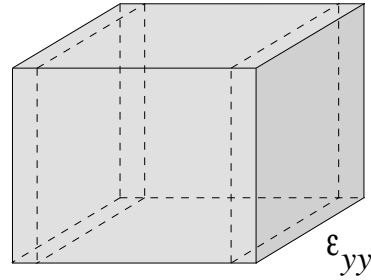
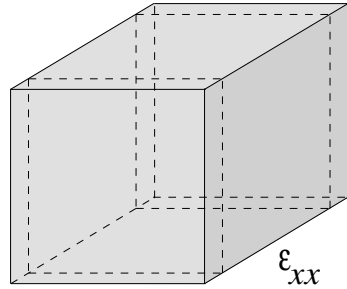


COLLEGE ONDERWERPEN

- 1 Spanningstensor
 - Spanningsdefinitie
 - Spanningstoestanden en voorbeelden
- 2 Rektensor
 - Relatieve verplaatsingen
 - Rekdefinities
 - Rektensor
- 3 Tensoreigenschappen
 - Introductie van tensoren
 - Transformatieregels
 - Cirkel van Mohr
 - Voorbeeld
- 4 Spannings-rek relatie
 - Cirkel van Mohr voor spanning en rek
 - Voorbeelden
- 5 Bezwijktoestanden
 - Spanningen in 3D
 - Von Mises en Tresca

SPANNINGEN





$$\epsilon_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial j} + \frac{\partial u_j}{\partial i} \right) \quad i, j = x, y, z$$

SPANNINGEN EN REKKEN

$$\boldsymbol{\varepsilon} = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{xy} & \varepsilon_{xz} \\ \varepsilon_{yx} & \varepsilon_{yy} & \varepsilon_{yz} \\ \varepsilon_{zx} & \varepsilon_{zy} & \varepsilon_{zz} \end{bmatrix} \quad \text{en} \quad \boldsymbol{\sigma} = \begin{bmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{bmatrix}$$

Verband tussen spanning en rek ?
4^e orde tensor (9x9=81 onbekende elementen)

SPANNINGEN EN REKKEN

$$\varepsilon = \left\{ \begin{matrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \\ \varepsilon_{xy} \\ \varepsilon_{yz} \\ \varepsilon_{zx} \end{matrix} \right\} \text{ of } \varepsilon = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{xy} & \varepsilon_{xz} \\ \varepsilon_{xy} & \varepsilon_{yy} & \varepsilon_{yz} \\ \varepsilon_{xz} & \varepsilon_{yz} & \varepsilon_{zz} \end{bmatrix}$$

6 REKKEN

$$\sigma = \left\{ \begin{matrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{matrix} \right\} \text{ of } \sigma = \begin{bmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} \end{bmatrix}$$

6 SPANNINGEN

MATERIAALGEDRAG

$6 \times 6 = 36$ onbekende elementen

matrix is symmetrisch $\gg$ 21 onbekende elementen

aeolotropie : het materiaal heeft in verschillende richtingen verschillende eigenschappen

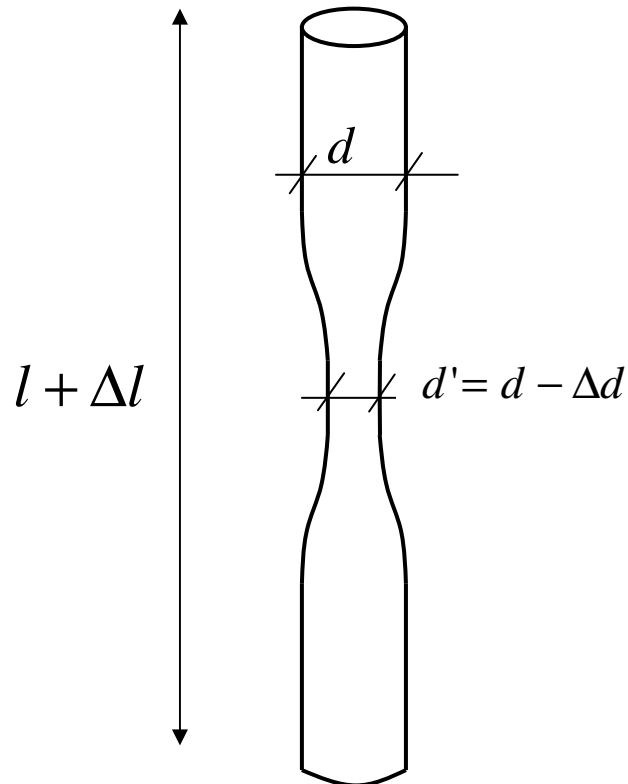
orthotropie : het materiaal heeft verschillende eigenschappen in 3 onderling loodrechte richtingen

anisotropie : aeolotropie en orthotropie

isotropie : het materiaal heeft in alle richtingen dezelfde eigenschappen

homogeen : het materiaal heeft in ieder punt dezelfde eigenschappen

REK = Specifieke verlenging



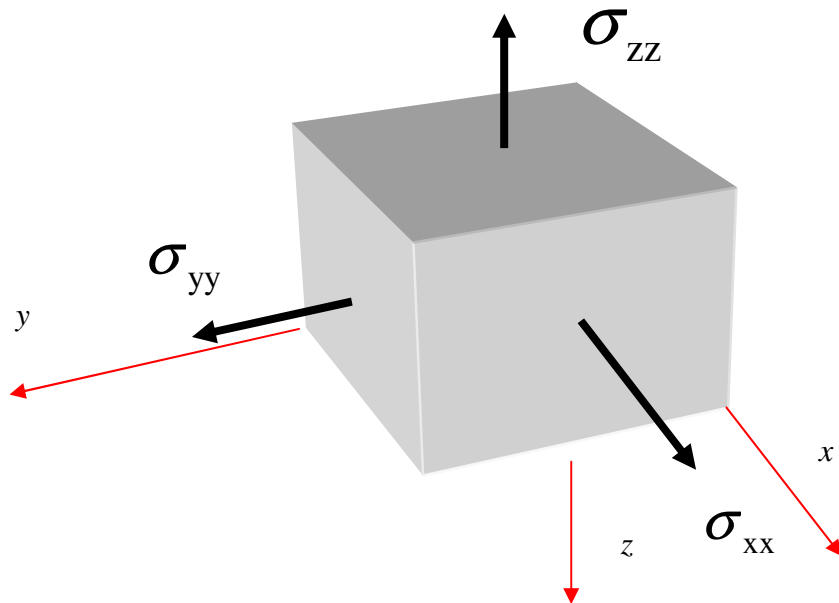
Lengte-richting : $\epsilon_l = \frac{\Delta l}{l}$

Dwars-richting : $\epsilon_d = \frac{\Delta d}{d}$

Insnoering : $\frac{\frac{\Delta d}{d}}{\frac{\Delta l}{l}} = \text{const} = \nu$

dwarscontractie-coëfficiënt of de constante van Poisson

VERLENGING DOOR TREK IN EEN RICHTING VEROORZAAKT EEN VERKORTING IN EEN RICHTING LOODRECHT DAAROP



Normaalrekken - normaalspanningen

$$\begin{aligned}\epsilon_{xx} &= \frac{\sigma_{xx}}{E} - \frac{\nu\sigma_{yy}}{E} - \frac{\nu\sigma_{zz}}{E} \\ \epsilon_{yy} &= -\frac{\nu\sigma_{xx}}{E} + \frac{\sigma_{yy}}{E} - \frac{\nu\sigma_{zz}}{E} \\ \epsilon_{zz} &= -\frac{\nu\sigma_{xx}}{E} - \frac{\nu\sigma_{yy}}{E} + \frac{\sigma_{zz}}{E}\end{aligned}$$

$$\begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \end{Bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu \\ -\nu & 1 & -\nu \\ -\nu & -\nu & 1 \end{bmatrix} \cdot \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \end{Bmatrix}$$

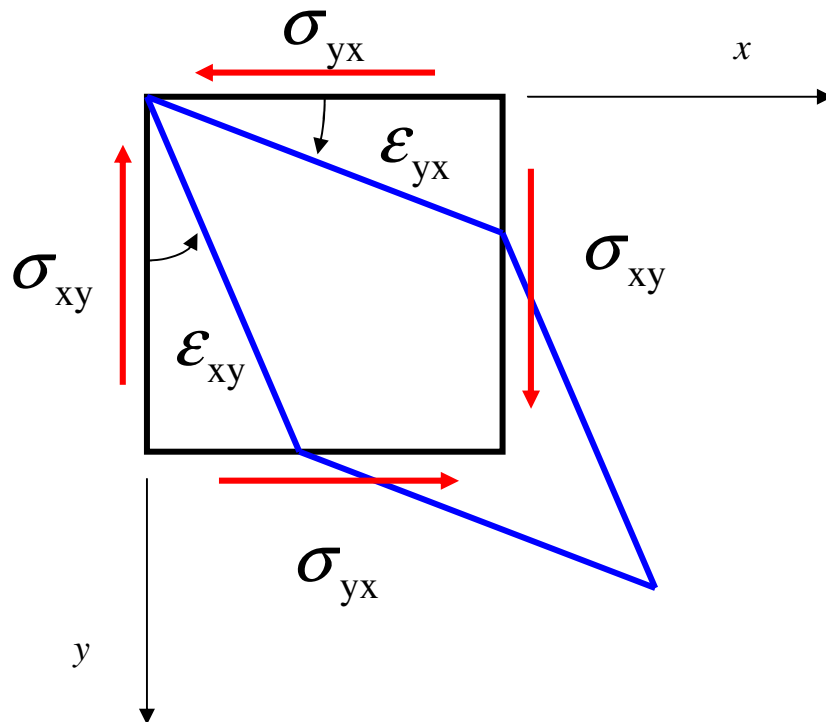
Afschuifhoek - schuifspanning

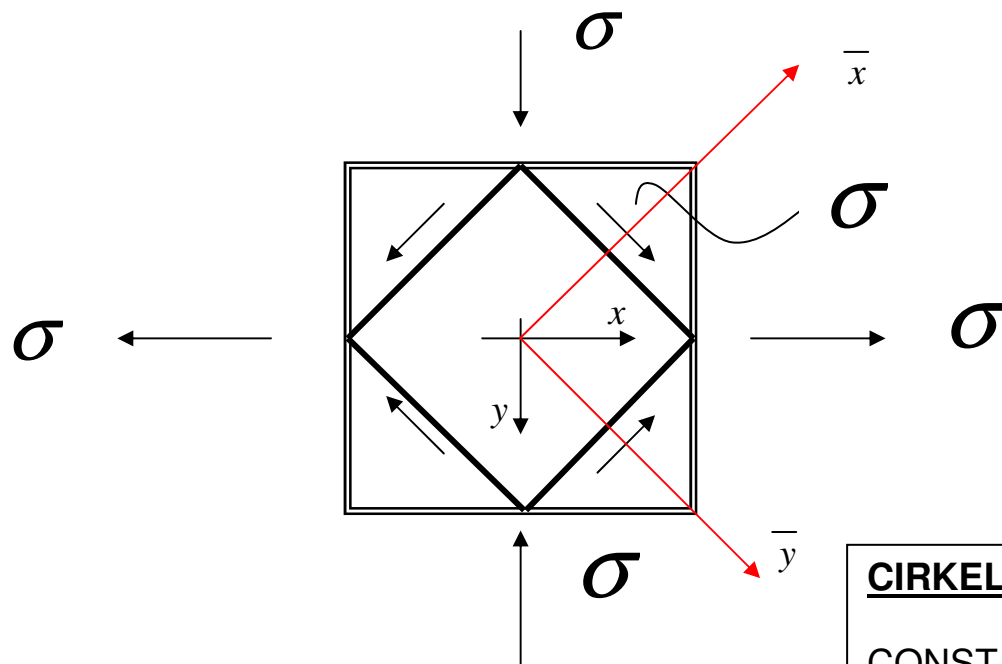
$$\gamma_{xy} = 2\varepsilon_{xy}$$

$$\sigma_{xy} = G\gamma_{xy}$$

dus :

$$2\varepsilon_{xy} = \frac{1}{G}\sigma_{xy}$$



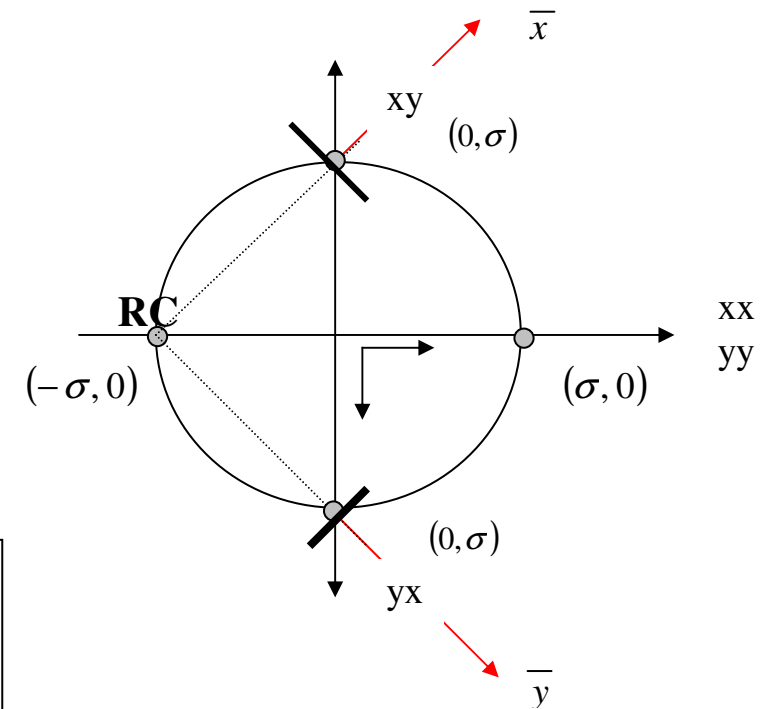


CIRKEL VAN MOHR

CONSTANTE
SCHUIFSPANNING OP
DE VLAKKEN ONDER 45
GRADEN

DUS :

ZUIVERE AFSCHUIVING



OORSPRONKELIJKE ASSENSTELSEL
(BUITENKANT VAN HET BLOK)

$$\sigma_{xx} = \sigma \quad \sigma_{xy} = 0$$

$$\sigma_{yy} = -\sigma \quad \sigma_{yx} = 0$$

dus :

$$\varepsilon_{xx} = \frac{1}{E} (\sigma_{xx} - \nu \sigma_{yy}) = \frac{1+\nu}{E} \sigma$$

$$\varepsilon_{yy} = \frac{1}{E} (\sigma_{yy} - \nu \sigma_{xx}) = -\frac{1+\nu}{E} \sigma$$

$$\varepsilon_{xy} = 0$$

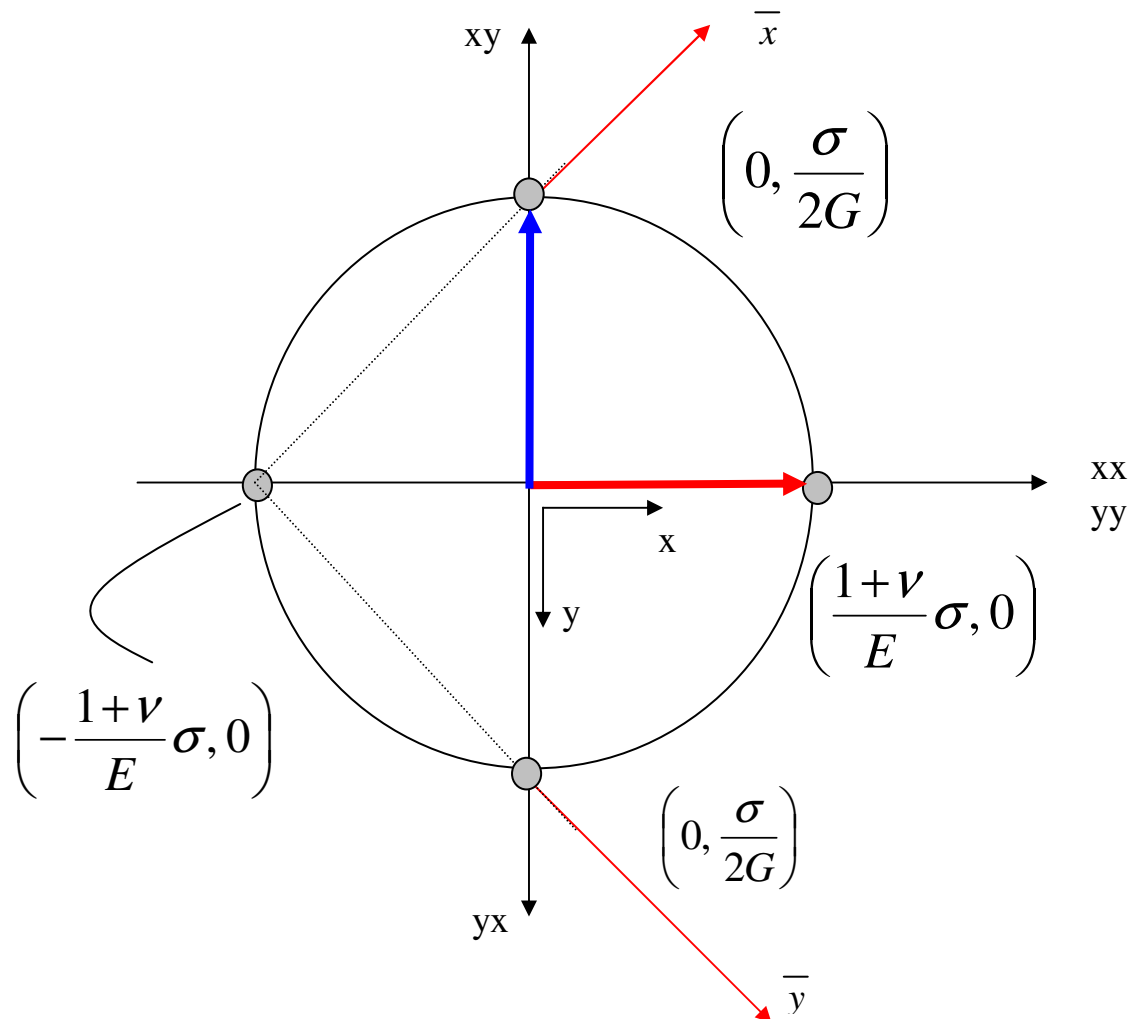
GEROTEERDE ASSENSTELSEL
(BLOKJE OP PURE AFSCHUIVING)

$$\varepsilon_{xx} = 0$$

$$\varepsilon_{yy} = 0$$

$$\varepsilon_{xy} = \frac{\sigma_{xy}}{2G} = \frac{\sigma}{2G}$$

$$\varepsilon_{yx} = \frac{\sigma_{yx}}{2G} = \frac{\sigma}{2G}$$



REKCIRKEL VAN MOHR

STRAAL VAN DE
CIRKEL IS UITERAARD
CONSTANT

$$\frac{1+\nu}{E} \sigma = \frac{\sigma}{2G}$$

$$G = \frac{E}{2(1+\nu)}$$

AFSCHUIVING

$$\sigma_{ij} = G\gamma_{ij} = G2\varepsilon_{ij} \quad \text{met: } i \neq j$$

$$G = \frac{E}{2(1+\nu)} \quad \Rightarrow \quad 2\varepsilon_{ij} = \frac{2(1+\nu)}{E} \sigma_{ij}$$

DUS :

$$\begin{Bmatrix} 2\varepsilon_{xy} \\ 2\varepsilon_{yz} \\ 2\varepsilon_{zx} \end{Bmatrix} = \frac{1}{E} \begin{bmatrix} 2(1+\nu) & 0 & 0 \\ 0 & 2(1+\nu) & 0 \\ 0 & 0 & 2(1+\nu) \end{bmatrix} \begin{Bmatrix} \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{Bmatrix}$$

ISOTROOP LINEAIR ELASTISCH MATERIAAL (WET VAN HOOKE)

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \\ 2\varepsilon_{xy} \\ 2\varepsilon_{yz} \\ 2\varepsilon_{zx} \end{Bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu & 0 & 0 & 0 \\ -\nu & 1 & -\nu & 0 & 0 & 0 \\ -\nu & -\nu & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\nu) \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{Bmatrix}$$

FLEXIBILITEITSRELATIE

ISOTROOP LINEAIR ELASTISCH MATERIAAL (WET VAN HOOKE)

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0 \\ \nu & 1-\nu & \nu & 0 & 0 & 0 \\ \nu & \nu & 1-\nu & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1-2\nu}{2} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1-2\nu}{2} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix} \begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \\ 2\varepsilon_{xy} \\ 2\varepsilon_{yz} \\ 2\varepsilon_{zx} \end{bmatrix}$$

positief voor

$$-1 < \nu < 0,5$$

STIJFHEIDSRELATIE

VLAKSPANNINGSTOESTAND (*b.v. x-y vlak*) ($\sigma_{zz} = 0$ $\sigma_{xz} = 0$ $\sigma_{zy} = 0$)

$$\varepsilon_{xx} = \frac{1}{E} (\sigma_{xx} - \nu \sigma_{yy})$$

$$\varepsilon_{yy} = \frac{1}{E} (\sigma_{yy} - \nu \sigma_{xx})$$

$$\varepsilon_{xy} = \frac{\sigma_{xy}}{2G}$$

$$\sigma_{xx} = \frac{E}{1-\nu^2} (\varepsilon_{xx} + \nu \varepsilon_{yy})$$

$$\sigma_{yy} = \frac{E}{1-\nu^2} (\varepsilon_{yy} + \nu \varepsilon_{xx})$$

$$\sigma_{xy} = 2G \varepsilon_{xy}$$

FLEXIBILITEIT

STIJFHEID

$$G = \frac{E}{2(1+\nu)}$$

VLAKE SPANNINGSTOESTAND (b.v. x-y vlak) $(\sigma_{zz} = 0 \quad \sigma_{xz} = 0 \quad \sigma_{zy} = 0)$

flexibiliteit :

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{xy} \end{Bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & 0 \\ -\nu & 1 & 0 \\ 0 & 0 & 1+\nu \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix}$$

stijfheid :

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & (1-\nu) \end{bmatrix} \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{xy} \end{Bmatrix}$$

HOOFDREKRICHTING VERSUS HOOFDSPANNINGSRICHTING

$$\tan 2\alpha_{\text{stress}} = \frac{\sigma_{xy}}{\frac{1}{2}(\sigma_{xx} - \sigma_{yy})}$$

$$\tan 2\alpha_{\text{strain}} = \frac{\varepsilon_{xy}}{\frac{1}{2}(\varepsilon_{xx} - \varepsilon_{yy})}$$

$$\varepsilon_{xx} = \frac{1}{E}(\sigma_{xx} - \nu\sigma_{yy})$$

$$\varepsilon_{yy} = \frac{1}{E}(\sigma_{yy} - \nu\sigma_{xx})$$

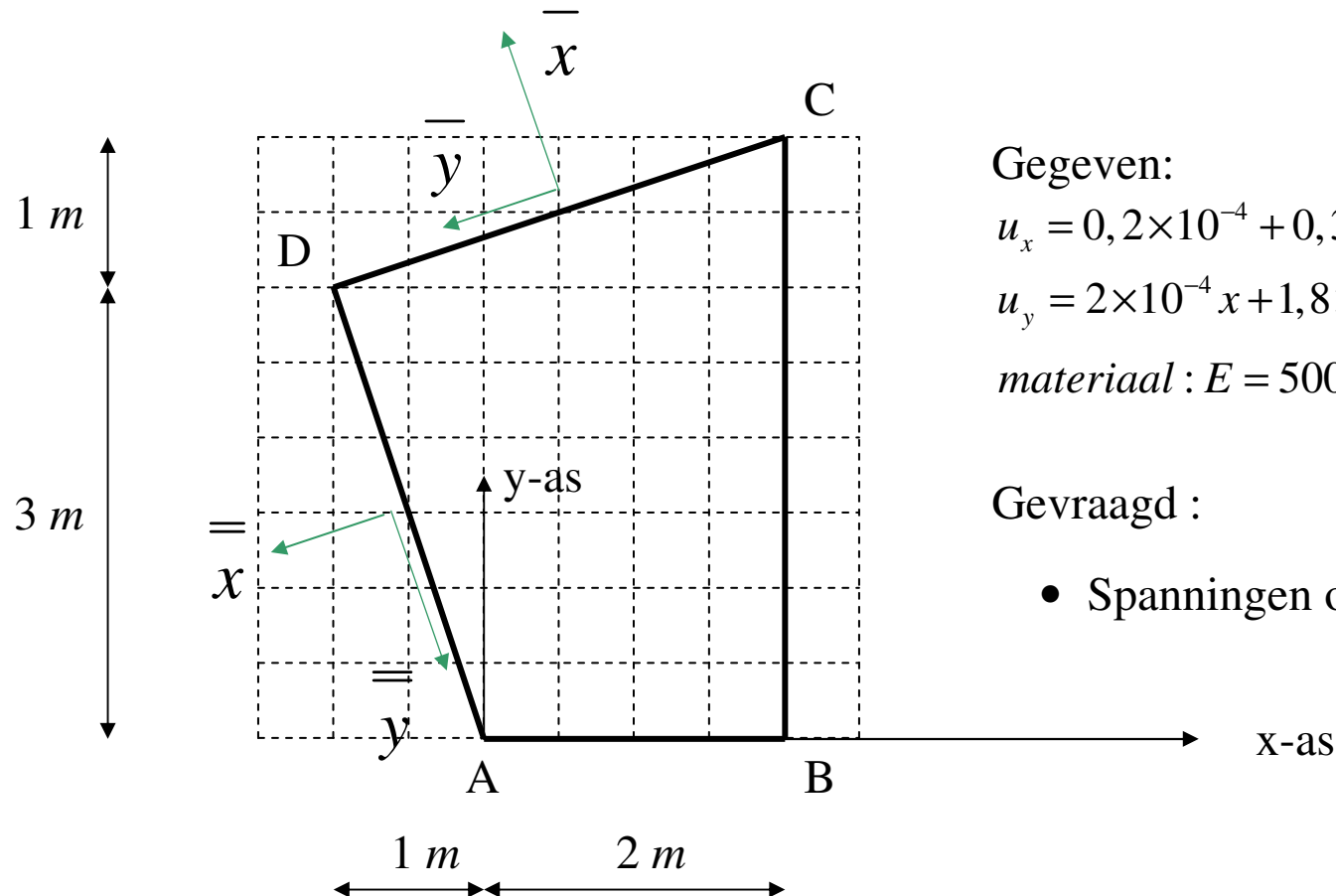
$$\varepsilon_{xy} = \frac{\sigma_{xy}}{2G}$$

$$G = \frac{E}{2(1+\nu)}$$

$$\tan 2\alpha_{\text{strain}} = \frac{\frac{\sigma_{xy}}{2G}}{\frac{1}{2E}((1+\nu)\sigma_{xx} - (1+\nu)\sigma_{yy})} = \frac{1}{2G} \times \frac{E}{1+\nu} \times \frac{\sigma_{xy}}{\frac{1}{2}(\sigma_{xx} - \sigma_{yy})} =$$

$$\frac{2(1+\nu)}{2E} \times \frac{E}{1+\nu} \times \frac{\sigma_{xy}}{\frac{1}{2}(\sigma_{xx} - \sigma_{yy})} = \frac{\sigma_{xy}}{\frac{1}{2}(\sigma_{xx} - \sigma_{yy})} = \tan 2\alpha_{\text{stress}}$$

VOORBEELD (zie ook rektensor vorig college)



Gegeven:

$$u_x = 0,2 \times 10^{-4} + 0,3 \times 10^{-4} x$$

$$u_y = 2 \times 10^{-4} x + 1,8 \times 10^{-4} y$$

materiaal : $E = 50000 \text{ N/mm}^2$; $\nu = 0,25$

Gevraagd :

- Spanningen op alle zijden

VAN REK NAAR SPANNING

Rekken

$$(\varepsilon_{xx}; \varepsilon_{xy}) = (0,3 \times 10^{-4}; 1,0 \times 10^{-4})$$

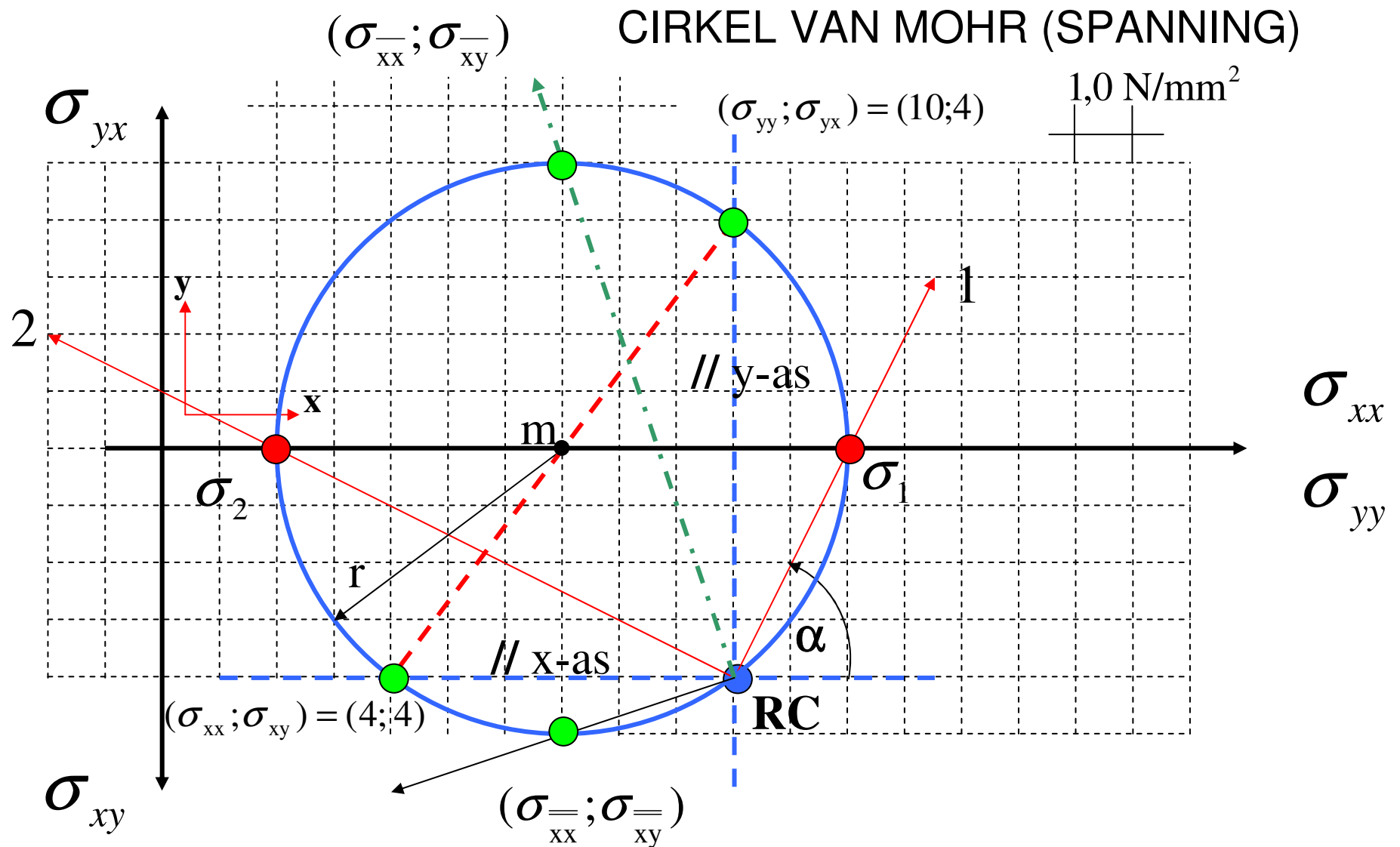
$$(\varepsilon_{yy}; \varepsilon_{yx}) = (1,8 \times 10^{-4}; 1,0 \times 10^{-4})$$

Spanning [N/mm²]

$$(\sigma_{xx}; \sigma_{xy}) = (4; 4)$$

$$(\sigma_{yy}; \sigma_{yx}) = (10; 4)$$

$$\begin{aligned}\sigma_{xx} &= \frac{E}{1-\nu^2} (\varepsilon_{xx} + \nu \varepsilon_{yy}) \\ \sigma_{yy} &= \frac{E}{1-\nu^2} (\varepsilon_{yy} + \nu \varepsilon_{xx}) \\ \sigma_{xy} &= \frac{E}{1+\nu} \varepsilon_{xy}\end{aligned}$$



VERGELIJK REKCIRKEL MET SPANNINGSCIRKEL

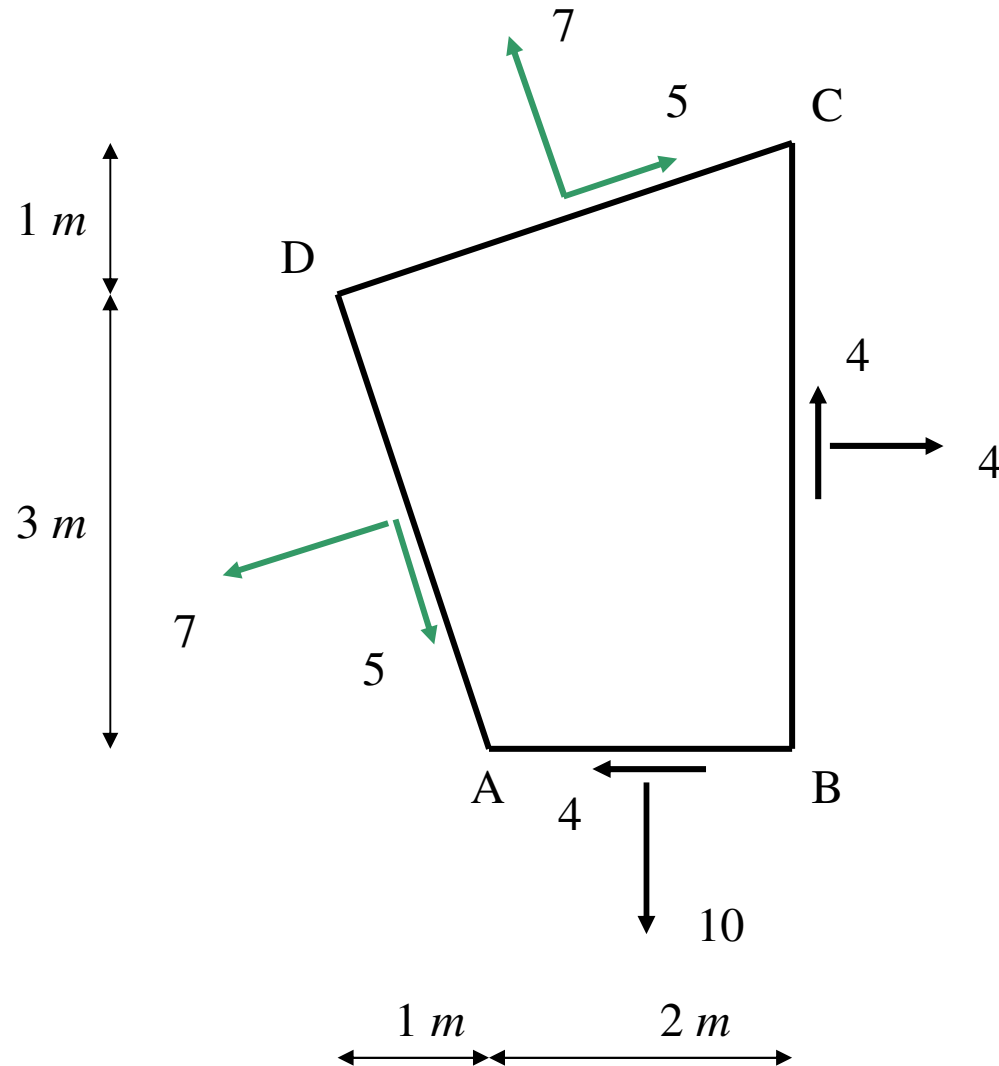
VOOR HOMOGEEN ISOTROOP MATERIAAL GELDT :

- HOOFDRICHTINGEN ZIJN GELIJK (zie bewijs blz 48)
- DUS PLAATS VAN HET R.C. IS IN BEIDE CIRKELS GELIJK

ALTERNATIEVE BEPALING SPANNINGSCIRKEL

- BEPAAL HOOFDSPANNINGEN UIT HOOFDREKKEN
- TEKEN SPANNINGSCIRKEL
- NEEM HOOFDRICHTINGEN (EN R.C.) UIT REKCIRKEL OVER

SPANNINGEN OP DE VLAKKEN [N/mm²]



- ZELF KRACHTEN EN MOMENTENEVENWICHT CONTROLEREN !